

**Blank Shear Machine Design
Sponsored by
The Boeing Company**

**Jim Bob Hudson
Phil Marquardt
Eric Myers
Amanda Taussig
boeing@cet.nau.edu**



**Spring 2001
Capstone Design
Northern Arizona University
College of Engineering and Technology**
<http://www.cse.nau.edu/Design/D4P/EGR486/ME/00-Projects/boeing/>

Abstract

This report details a new design for a specialized “Chaku-Chaku” prototype sponsored and to be used by The Boeing Company. The machine will be integrated into a single part flow manufacturing line at the Light Structure Fabrication and Assembly division in Auburn, WA. This Machine will enable Boeing to manufacture blanks that were previously purchased from an outside vendor. Not only can the company produce the standard 4” x 6” blank, they will also be able to cut a variety of sizes. This will all be accomplished in one compact “Chaku-Chaku” machine.

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Introduction

The Boeing Company utilizes a process termed “Chaku-Chaku”, which means, “load-load” in their manufacturing process of small air frame parts. This term refers to the single part flow process when the operator proceeds from machine to machine, taking a part from the previous operation and loading it in the next machine until the final part to be handled is the finished component. The “Chaku-Chaku” single part process increases the amount of completed parts in an allotted time. Each Chaku-Chaku line is comprised of approximately 5 relatively small machines, all run by one operator.

Until this time, Boeing has out-sourced its aluminum blanking. The Boeing Company uses a standard 4” x 6” blank for all their light structure Chaku-Chaku machines. This leads to excess of material after the parts are punched from the blank. From the visit in October, the team noticed an approximate 80% waste in material for a certain geometry that was being produced. The production of the parts has, up until now has been a source of inefficiency and material waste.

The Boeing Company has asked a team of Northern Arizona University students to create a Chaku-Chaku that would cut aluminum in to lengths specified by the user. The machine would then be integrated into the current Chaku-Chaku lines in the Light Structures Fabrication and Assembly plant in Auburn, WA.

Scope of Work

The Boeing Company has asked for a finished prototype that meets or exceeds their specifications. In addition to the finished prototype, they have requested an itemized budget, design drawings/decisions, detailed drawings, appropriate mathematical analysis, project schedule, detailed budget, maintenance and calibration.

Specifications and Requirements

The Boeing Company provided a list of requirements and specifications to be taken into consideration when designing and building the blank shear machine.

The requirements includes:

- Aluminum will be delivered to the machine from a coil or from 4' strips.
- The machine will be powered by 120VAC 15A and/or 80psi shop air.
- There will be no marring on the blank's surface.
- The machine will deliver one blank within 30 seconds.
- The machine must be portable.
- The machine shall have the dimensions: 4ft high by 4ft deep by 2ft wide.
- The machine will straighten raw material from a coil, before being processed.



The blanks also have a list of specifications as follows:

- Material thickness
0.028" – 0.100"
- Material width
2" – 12"
- Material length
2" – 12" / 0.125" increments
0.03" tolerance
- Edge Chamfer
0.0625"
- Edge Burr
0.01" max
- Blank Bending
0.1" max

Design Philosophy

An aggressive design philosophy is necessary in order to produce a satisfactory design. An underlying blueprint of the design for the blank coil shear machine was devised to guide in the decision making process.

The design philosophy includes:

- The machine will be designed for the functionality, reliability and quality of the prototype.
- The machine will be simple to use.
- The machine will be safe.
- The machine will minimize time and effort to load, adjust and operate.
- Minimal maintenance will be needed on the machine.
- The prototype will function reliably throughout machine life.

Design decisions

When considering possible designs, an ample amount of research was conducted. The research was divided into the separate systems in the shear Chaku-Chaku. These were blank cutting, coil straightening, and automation.

For blank cutting, it was necessary to research an angled guillotine knife, a flat knife, pneumatic nibblers, a band saw, a laser knife, a water knife, and a slitter knife. In order to evaluate what was the best option to cut the blank, it was necessary to consider price, the amount of force needed, the plausibility of integration into the machine and the ease of use of the particular device. The flat knife would require a great amount of force to cut the material. Nibblers would make a very jagged edge on the blank. A band saw would be too difficult to integrate into the machine. Laser and water knives were too expensive and impractical. The slitter knife would have been difficult to use. Therefore it was concluded that the angled guillotine knife would best suite the needs of the project because the force to cut a piece would be greatly reduced, and the knife would be easy to integrate into our compact machine.

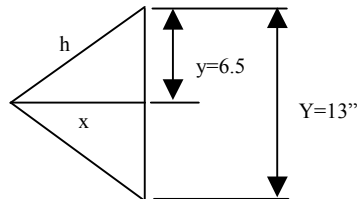
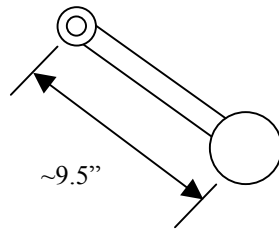
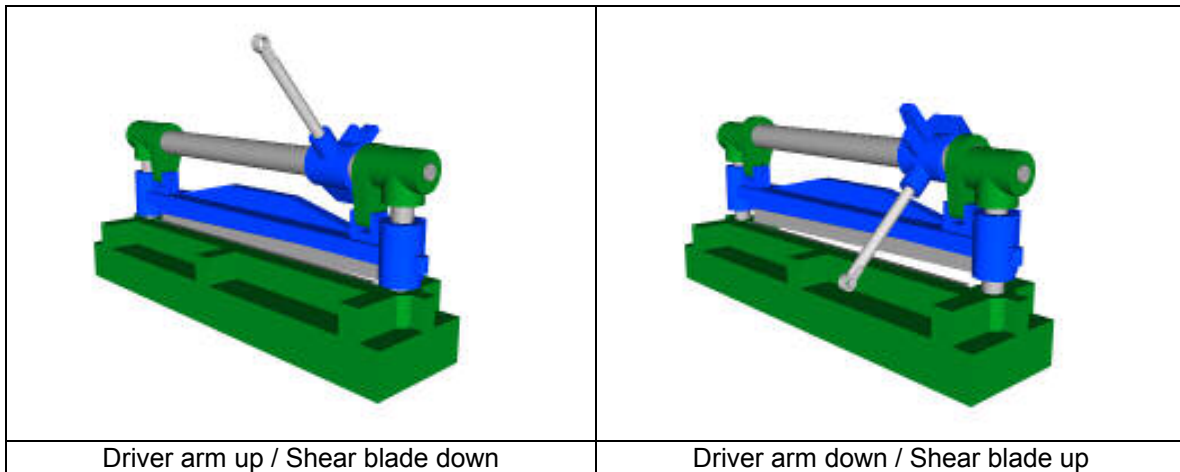
The next sub-system to be evaluated was the coil straightener. The team researched various sheet metal straightening devices. Some options that we considered were reverse rolling, pressing and tensioning. The team concluded that both pressing and tensioning would have a tendency to mar the blank and perhaps undesirably alter the material properties. In addition, both of these options would unnecessarily complicate the design. Reverse rolling is most often used for straightening wire and was found to be the best alternative for the need of the project. Reverse rolling is commonly used in many applications, and integrated well into our design.

The next area of research was how to automate the indexing system. The material, as per the requirements, would be cut in some increment of $1/8$ ". Viable indexing options that were researched were: optical sensors, adjustable stops, edge locators, stepper motors, and rotary sensors. An optical sensor was the best choice. In consideration for the electronics the optical sensor was the easiest to integrate, and was therefore the best choice.

With the major design decisions in place, there was a struggle to decide between building a shear and purchasing one. Research indicated a hand operated shear closely fit our design requirements. A similar hand shear in the machine shop showed promising results before the decision was made to purchase our own shear. Since the test was successful, the Di-Acro hand shear was purchased and incorporated into the machine design. This meant obtaining permission from the "Chaku-Chaku" group to exceed the specified width since the purchased shear exceeded the requirements by approximately 8".

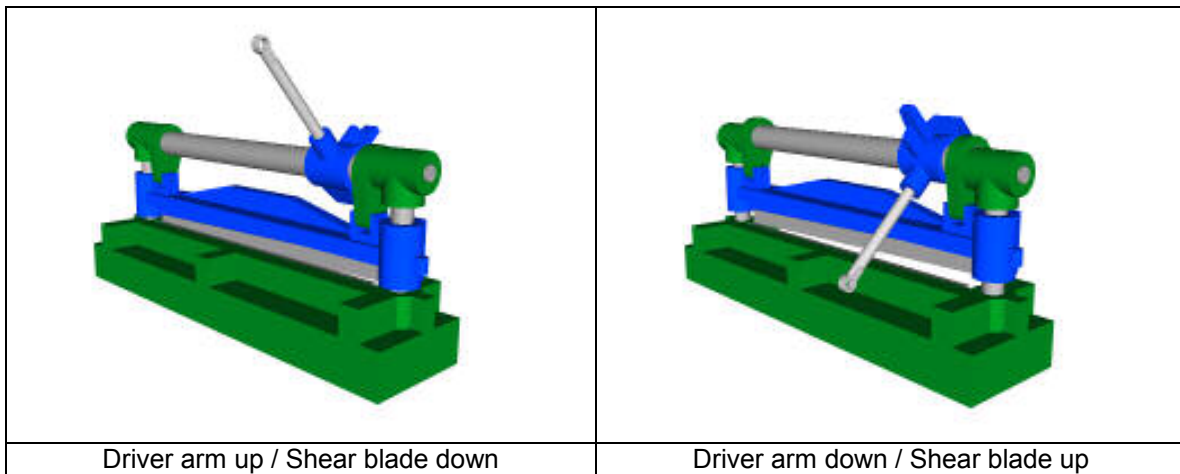
Supporting mathematical models

Range of Motion



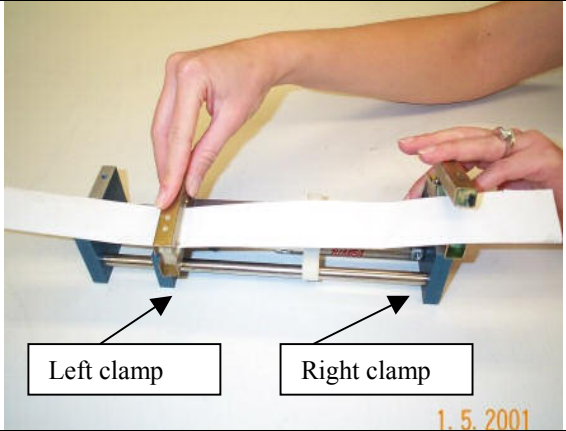
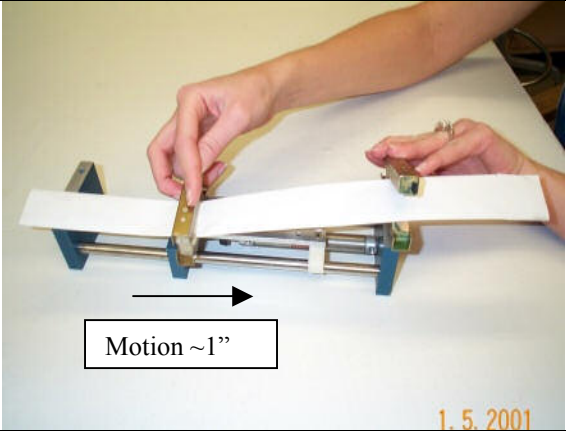
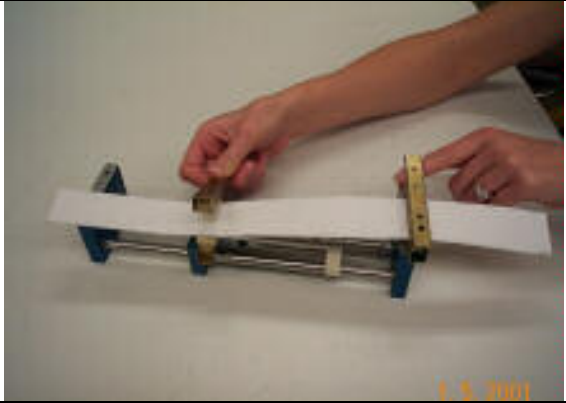
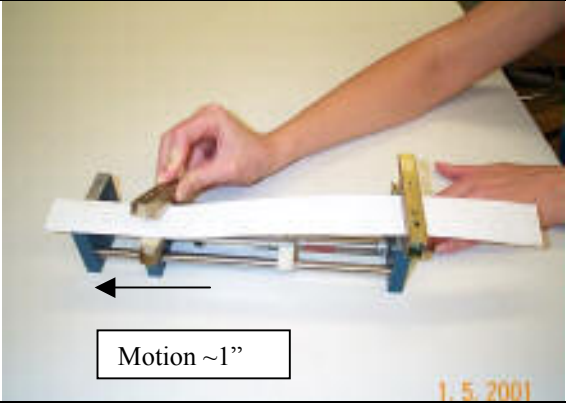
<p> $y := 6.5 \text{ in}$ $h := 9.5 \text{ in}$ </p> <p> $x := \sqrt{h^2 - y^2}$ </p> <p> $x = 6.928 \text{ in}$ approximately 7" </p> <p> $\theta := \sin\left(\frac{h}{y}\right)$ </p> <p> $\theta = 56.954 \text{ deg}$ </p> <p> $\omega := 2 \cdot \theta$ </p> <p> $\omega = 113.908 \text{ deg}$ </p>	<p>Range of motion analysis</p> <p>In choosing an air cylinder a few critical dimensions were required.</p> <p>The distance x is critical in determining if any components would be in the way during mid cycle of air cylinder.</p> <p>Angle ω is important in determining the full range of motion of the shear knife.</p>
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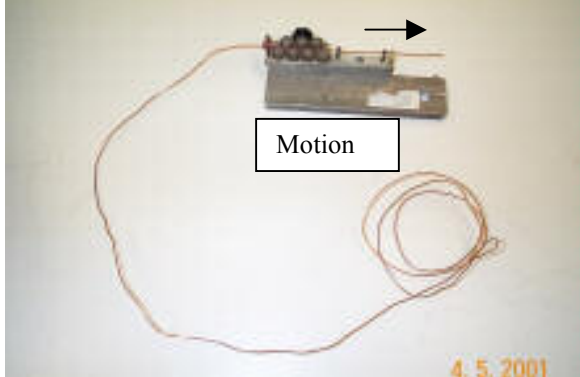
Force to Shear



$y_{\text{arm}} := 13 \text{ in}$	Total distance moved by shear arm	Available Force to Shear The calculated maximum force available for shearing at our 80-psi design specification.
$y_{\text{blade}} := 0.5 \text{ in}$	Total distance moved by shear blade	
$\frac{y_{\text{arm}}}{y_{\text{blade}}} = 26$	Shear Mechanical Advantage	
$P_{\text{max}} := 80 \cdot \text{psi}$	Maximum operating pressure	
$d_{\text{piston}} := 3.25 \cdot \text{in}$	Cylinder piston diameter	
$A := \frac{\pi}{4} \cdot d_{\text{piston}}^2$		
$A = 8.296 \cdot \text{in}^2$	Piston working area	
$F_{\text{arm}} := P_{\text{max}} \cdot A$		
$F_{\text{arm}} = 663.661 \cdot \text{lbf}$	Maximum lever force	
$F_{\text{shear}} := \frac{y_{\text{arm}}}{y_{\text{blade}}} \cdot F_{\text{arm}}$		
$F_{\text{shear}} = 1.726 \cdot 10^4 \cdot \text{lbf}$	~ 17,260 lbf Maximum shearing force	

Physical modeling


Slide 1	Slide 2
	
<ul style="list-style-type: none"> • Right clamp simulates shearing action • Left clamp simulates pneumatic clamp mechanism • Air cylinder (underneath paper) simulates linear actuation • White paper simulates aluminum strips 	<ul style="list-style-type: none"> • Notice paper is approximately 1" longer after actuation has occurred • Notice clamps are still in original positions
Slide 3	Slide 4
	
<ul style="list-style-type: none"> • Right clamp shear closes and simulates material being cut • Left clamp opens 	<ul style="list-style-type: none"> • Reverse linear actuation occurs which causes the clamp to come back for new material • Left clamp closes on a new section of material • Right clamp closes and machine is ready for new cycle


Slide 1	Slide 2
	
<ul style="list-style-type: none"> • Notice bent copper wire 	<ul style="list-style-type: none"> • Bent material is being feed into wire straightening device
Slide 3	Slide 4
	
<ul style="list-style-type: none"> • As you can see material is reversed rolled causing the material to yield straight 	<ul style="list-style-type: none"> • Material can be continuously be straightened by pulling it through rollers

Operation


To Load the machine place the LOAD/RUN toggle to LOAD. Place the material under the feed clamp. Adjust guides as needed. For use with a coil, set the guides loosely and square the material by adjusting the position of the coil stand. Switch toggle back to RUN


To adjust the length of the blank enter the length in $1/8^{\text{th}}$ of an inch:

Press  to select preset 2 (P2)

Press  to select the digit

Press  to adjust value

Press  to save the value

Press  to preset the counter

To adjust the rollers:

Remove front pins and lift the top frame to load material.

Close the top frame and reinsert front pins.

Adjust roller height so that material is flat.

Lower front adjustment by $.5 * y_{\text{max}}$ (see table on next page)

Lower rear adjustment by $1.5 * y_{\text{max}}$ (one full turn = $1/8$ inch)



$$y_{\max} := \frac{F \cdot l^3}{48 \cdot E \cdot I}$$

$$E_{\text{total}} := E_{\text{modulus}} \cdot (1 + \nu)$$

$$I := \frac{b \cdot h^3}{12}$$

	t	E	I'	Sy	M'	F'	y _{max}
2011-T6	0.02	10.3	6.67E-07	24.5	1.63	0.59	0.225
	0.03	10.3	2.25E-06	24.5	3.68	1.34	0.150
	0.04	10.3	5.33E-06	24.5	6.53	2.38	0.112
	0.05	10.3	1.04E-05	24.5	10.21	3.71	0.090
	0.06	10.3	1.80E-05	24.5	14.70	5.35	0.075
	0.07	10.3	2.86E-05	24.5	20.01	7.28	0.064
	0.08	10.3	4.27E-05	24.5	26.13	9.50	0.056
	0.09	10.3	6.08E-05	24.5	33.08	12.03	0.050
	0.1	10.3	8.33E-05	24.5	40.83	14.85	0.045
2024-T4	0.02	10.3	6.67E-07	43	2.87	1.04	0.394
	0.03	10.3	2.25E-06	43	6.45	2.35	0.263
	0.04	10.3	5.33E-06	43	11.47	4.17	0.197
	0.05	10.3	1.04E-05	43	17.92	6.52	0.158
	0.06	10.3	1.80E-05	43	25.80	9.38	0.131
	0.07	10.3	2.86E-05	43	35.12	12.77	0.113
	0.08	10.3	4.27E-05	43	45.87	16.68	0.099
	0.09	10.3	6.08E-05	43	58.05	21.11	0.088
	0.1	10.3	8.33E-05	43	71.67	26.06	0.079
7075-T6	0.02	10.3	6.67E-07	78.6	5.24	1.91	0.721
	0.03	10.3	2.25E-06	78.6	11.79	4.29	0.481
	0.04	10.3	5.33E-06	78.6	20.96	7.62	0.361
	0.05	10.3	1.04E-05	78.6	32.75	11.91	0.288
	0.06	10.3	1.80E-05	78.6	47.16	17.15	0.240
	0.07	10.3	2.86E-05	78.6	64.19	23.34	0.206
	0.08	10.3	4.27E-05	78.6	83.84	30.49	0.180
	0.09	10.3	6.08E-05	78.6	106.11	38.59	0.160
	0.1	10.3	8.33E-05	78.6	131.00	47.64	0.144

(t = thickness (in), E = modulus of elasticity (Mpsi), I' = moment of inertia (in⁴) / per unit width, Sy = yield strength (kpsi), M' = moment / per unit width (lbf*in), F = force / per unit width (lbf), Ymax = max deflection from offset rollers (in))

Final project schedule

Start Date	End Date	Task
10/30/00	4/20/01	Web Page
1/08/01	3/26/01	Design
1/08/01	3/26/01	Drawings
3/13/01	3/13/01	Design Review Pres.
3/16/01	3/16/01	Design Review Rpt.
3/19/01	3/29/01	Component Integration
3/29/01	3/29/01	Status report #3
3/29/01	4/3/01	Testing
4/3/01	4/10/01	Prototype Refinement
4/11/01	4/17/01	Presentation Preparation
4/17/01	4/17/01	Peer Presentation
4/20/01	4/20/01	Final Presentation
4/23/01	5/4/01	Report Presentation
5/4/01	5/4/01	Final Report Due

For an more detailed schedule please visit our website and look in the calendar section;
<http://www.cse.nau.edu/Design/D4P/EGR486/ME/00-Projects/boeing/>

Conclusion

The working prototype of the Chaku-Chaku shear machine has met the specifications provided by The Boeing Company with a notable exception of external dimensions. The prototype performs up to the design specification. In searching for a shear to closely meet our requirements, we located a 12" Di-Acro shear. This 12" hand shear would have been a better fit into our design however, the new cost of this shear would have used our entire budget. In further research we found a 24" used Di-Acro shear that fit well within the budget and was identical to the 12" shear except for the width. The 24" used shear required minimal modification and setup. This shear will provide Boeing a long life of operation. Replacement parts can be purchased directly from Di-Acro (See reference section).

The one feature that we were unable to fully test is the straightening system. There is evidence that the straightening rollers can indeed yield material, however the team was unable to feed aluminum from a coil into the machine.

Overall, the team feels that this project was highly successful. It is not very often that a prototype works correctly the first time, but this prototype performed exceptional without any major modification. We feel grateful for the opportunity to have participated in this exciting project for The Boeing Company. Please see Appendix A for further redesign considerations and Appendix E for a detailed project budget.

Project Special Thanks

Dick Radcliff - Hydraulics Expert

Greg Thomas - owner: The Thomas Co.

Ron Boyer - Boyer Metal

Dave Noack - East Flagstaff

Kirk McDowell - ACC Machinery, Phx.

Brian Burt - Mesa Machinery Sales

George Bain - W.L. Gore

Dave Myers - W.L Gore

Dave Hartman - Project Supervisor, technical advisor

Don McCallum - Shop foreman, technical advisor

Justin Myers - Shop tech, technical advisor

We would like to extend our greatest thanks to a few people that took time out of their busy live to help us. This project would not have been a success without their input.

References

Chaku-Chaku Research

- “The Superfactory Project is a compilation of links supporting innovative manufacturing...”
<http://www.superfactory.com/>
- Lean Manufacturing glossary of terms
<http://www.rec-communications.com/lean/glossary.htm>

Guilbtine Knife research

- Hand operated guillotine knives
<http://www.diacro.com/>
PO Box 9700
Canton, OH 44711
(330) 499-3172

New/Used Machinery

- Company where used Di-Acro was purchased
ACC Machinery Company Inc, 747 Grand Ave, Phoenix, AZ 85007
(602) 258-7330
- Brian Burt company president was helpful in discovering new resources.
Mesa Machinery Sales, Inc
<http://www.mesamachinery.com>
128 W. Boxelder Pl., Suite 117, Chandler, AZ 85225
(480) 545-0275
(800) 542-8559

Text Reference

- Shigley, Joseph Edward. Mechanical Engineering Design 5th Edition, 1989
McGraw-Hill, Inc

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